Amendments to the Specificaiton

Amend paragraph [0033], last line, as follows.

[0033] Fig. 8 (a) is a graph of the x_2 amplitudes as a function of driving frequency for the dual mass oscillator showing the effect of variations in mass ratio μ_x = ~~(m2 + m3) = m1~~ $(m_2 + m_3)/m_1$.

Amend paragraph [0103], third line, as follows.

[0103] The degree of mechanical amplification depends on the ratio of the resonance frequencies of the isolated active system 38 and passive mass-spring system 40, namely $\gamma_x = \omega_{2x} / \omega_{1x}$ ~~ω_{2x}~~ $= (k_{2x}m_1/k_{1x}(m_2 + m_3))^{1/2}$. The optimal frequency ratio γ_x has to be determined such that γ_x is high enough for high mechanical amplification, and high oscillation amplitudes of passive mass as illustrated in the graph of Fig. 8(b). From the optimal values of μ_x and ω_{1x}, the drive direction spring constant k_{1x} of the active mass 16 is obtained. Finally, the damping conditions of the overall device have to be checked to verify that damping values are in the region where the response gain in the antiresonance region is insensitive to damping variations as illustrated by the graph of Fig. 9.